

Modeling, Validation and Physics of Turbulent Flows: opportunities offered by petascale Direct Simulation

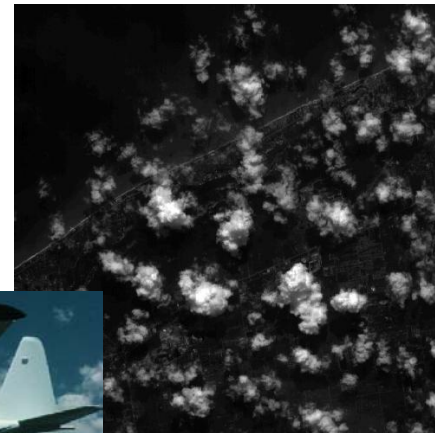
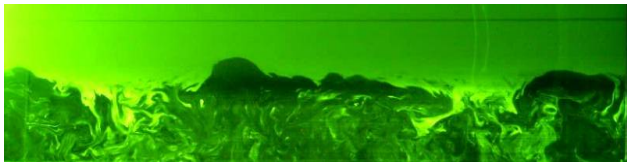
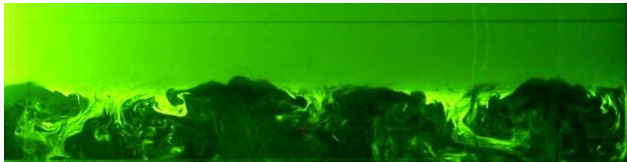
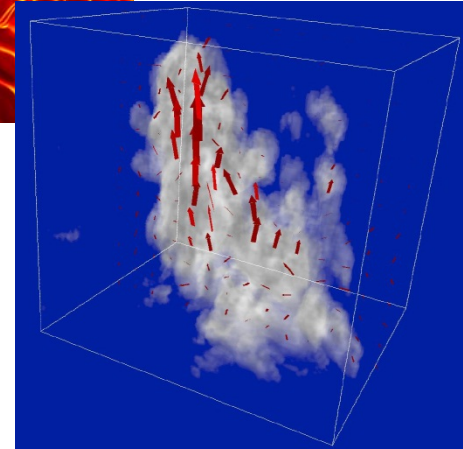
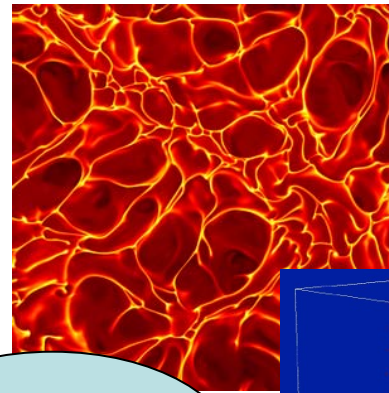
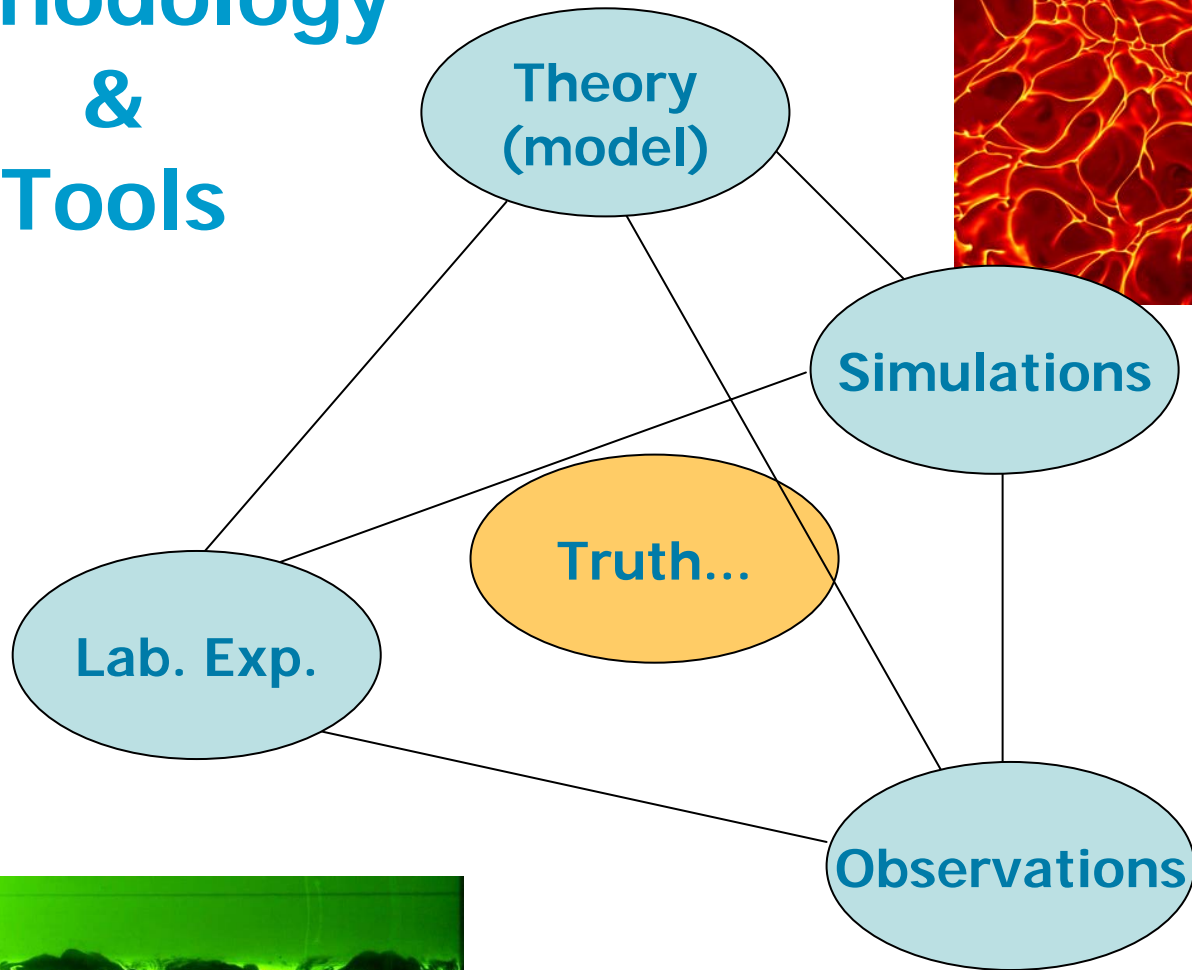


Harm Jonker (Multi-Scale Physics, Delft, Netherlands)

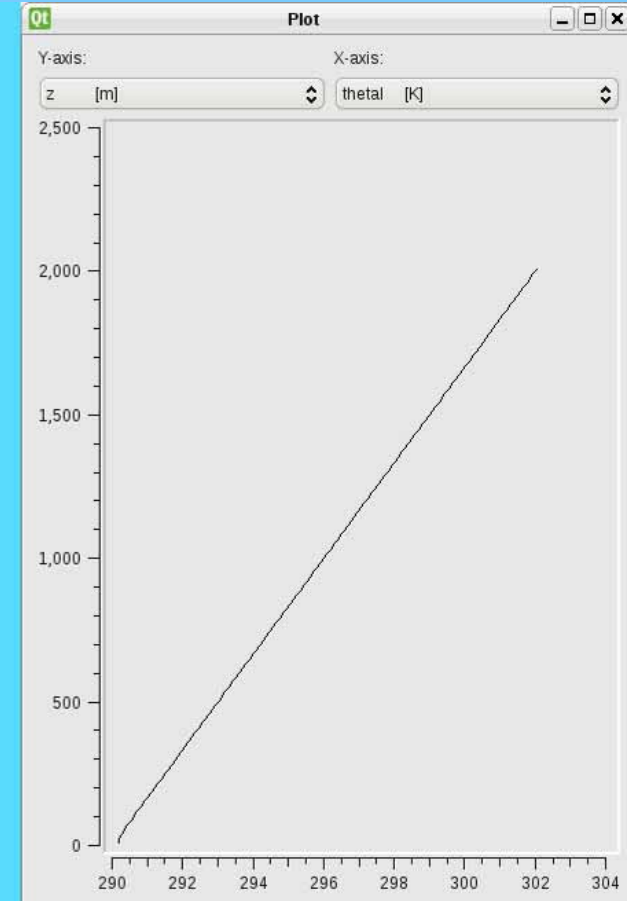
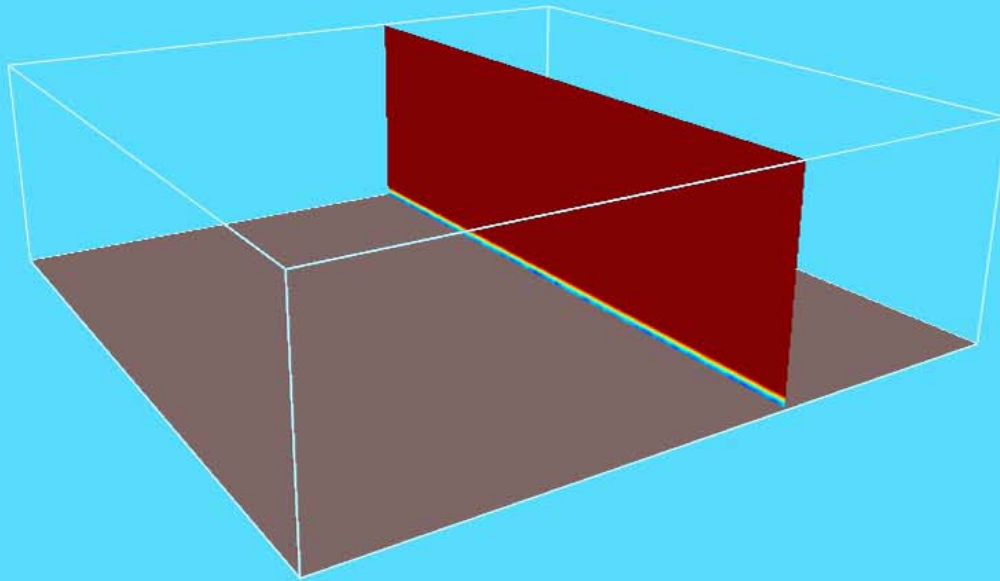
Peter Sullivan & Ned Patton (National Center for Atmospheric Research, USA)

Maarten van Reeuwijk (Imperial College, UK), Jerome Schalkwijk (Delft)

Methodology & Tools



Burgers – conference 2000

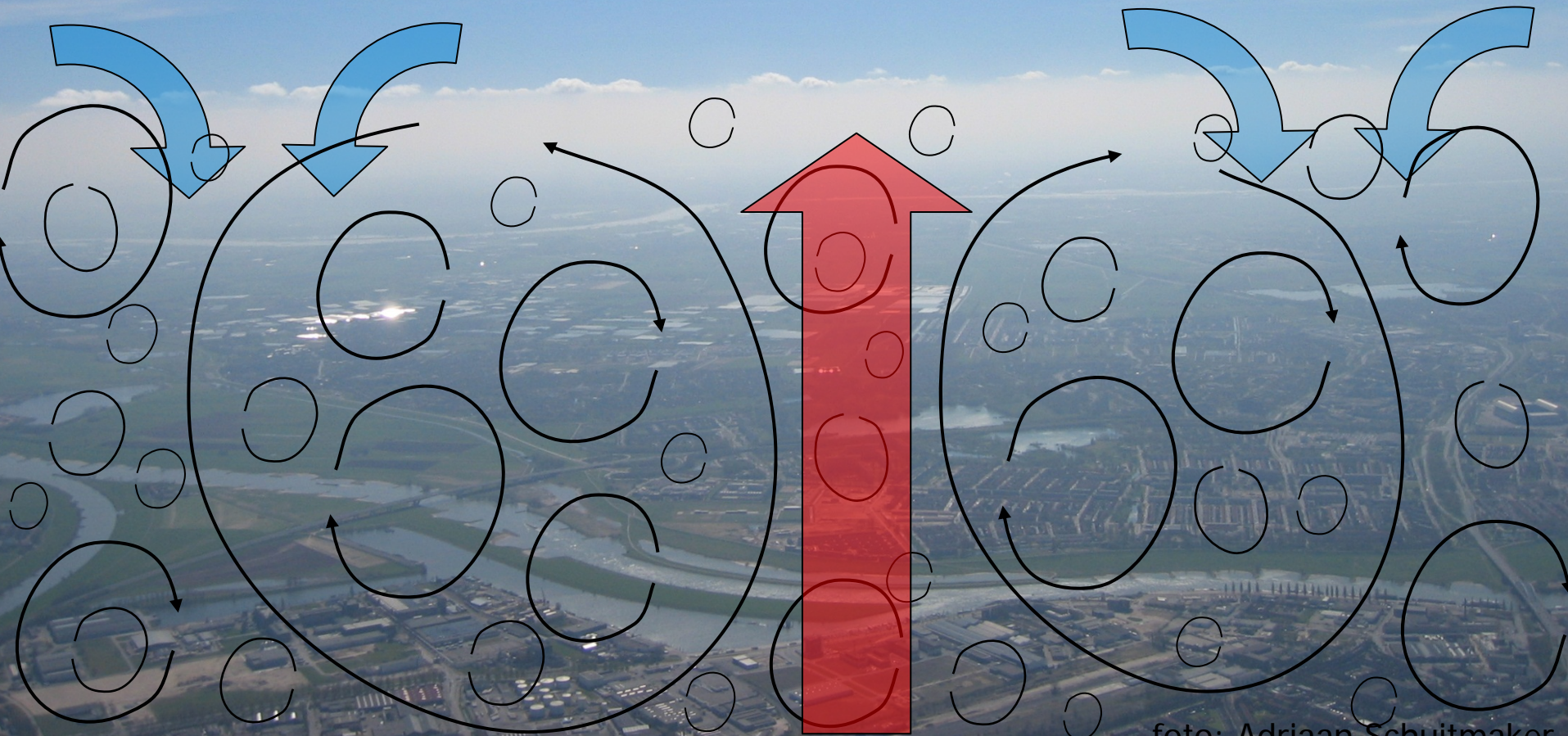


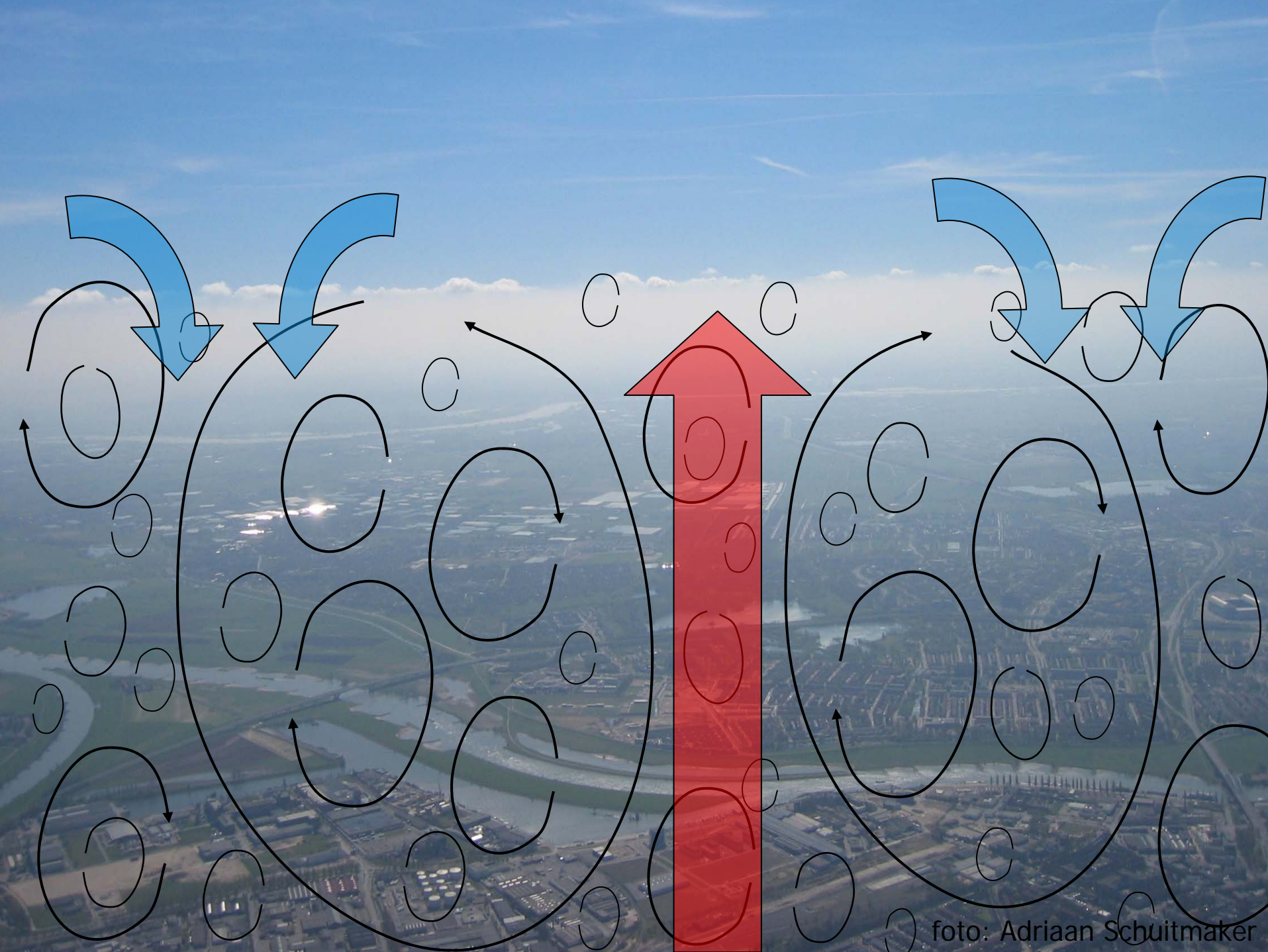
$\theta \rightarrow$

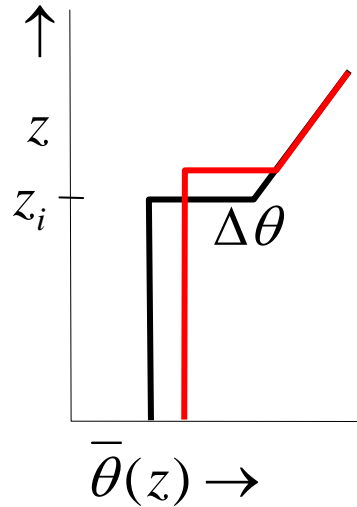
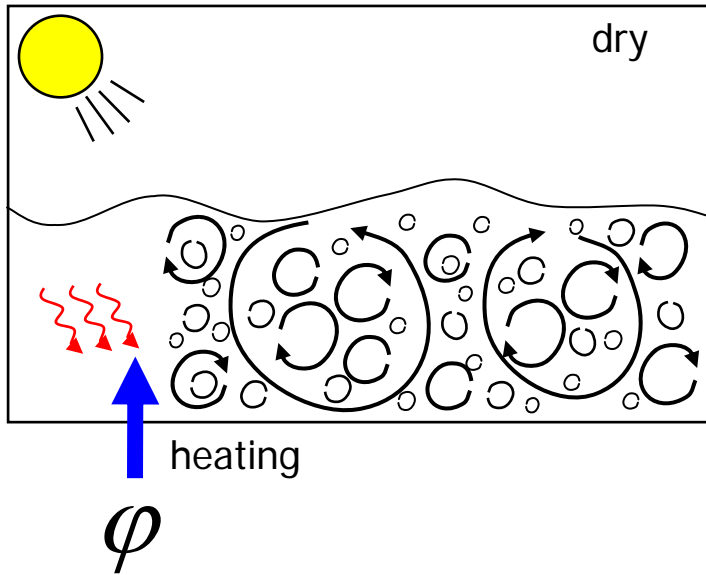
Large Eddy Simulation (GPU)

courtesy Jerome Schalkwijk

The convective boundary layer ...







growth-rate:

$$w_e = \frac{dz_i}{dt}$$

'law'

$$w_* = \left(\frac{g}{\theta_0} \varphi z_i \right)^{1/3}, \quad \text{Ri} = \frac{g}{\theta_0} \frac{\Delta\theta z_i}{w_*^2}$$

$$\frac{w_e}{w_*} = A \text{Ri}^{-1}$$

$$A = 0.2$$

Water Tank: Deardorff, Willis and Stockton, JFM 1980

Laboratory Experiments (thermal convection tank)

Deardorff et al. water tank model of shear-free CBL (1960-80s)

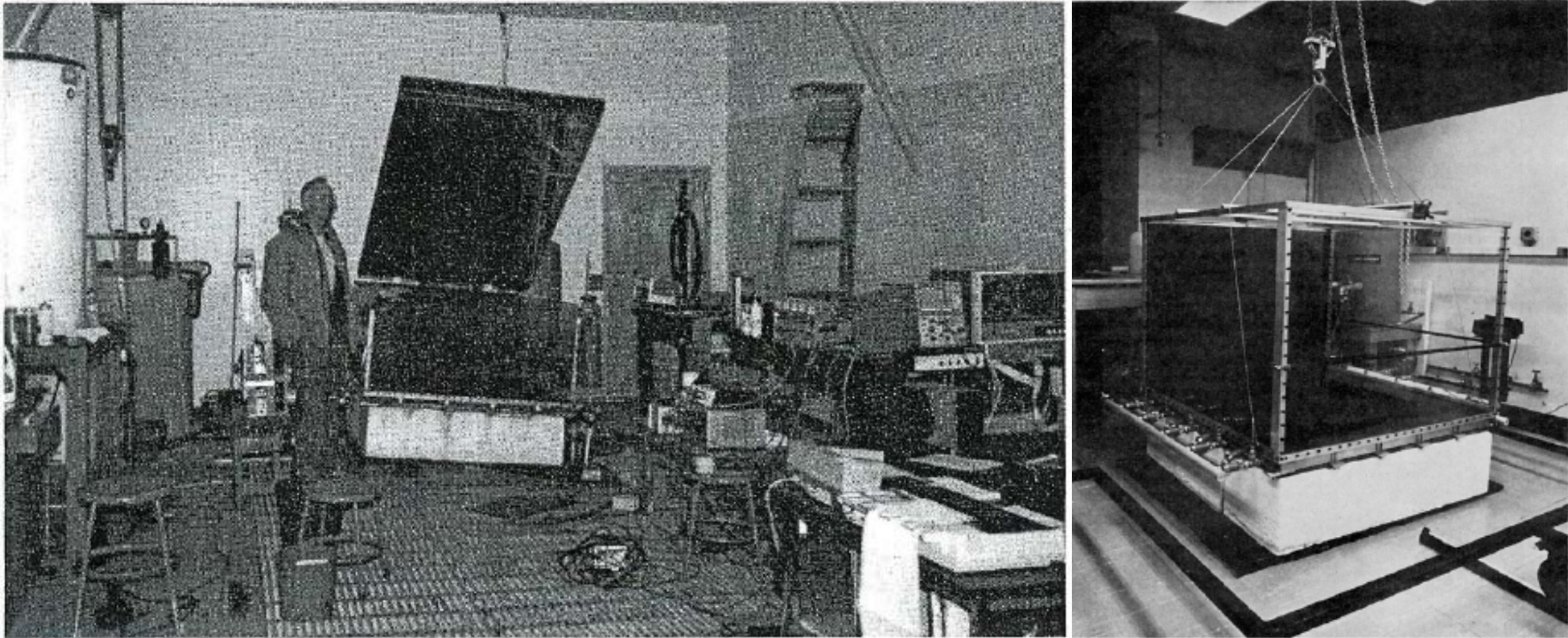


Figure 7.10 Tank used by Deardorff (pictured above) and Willis in their laboratory experiments. (Deardorff and Willis.)

From Sorbjan, Z., 1989: *Structure of the Atmospheric Boundary Layer*, Prentice Hall, 317 pp.

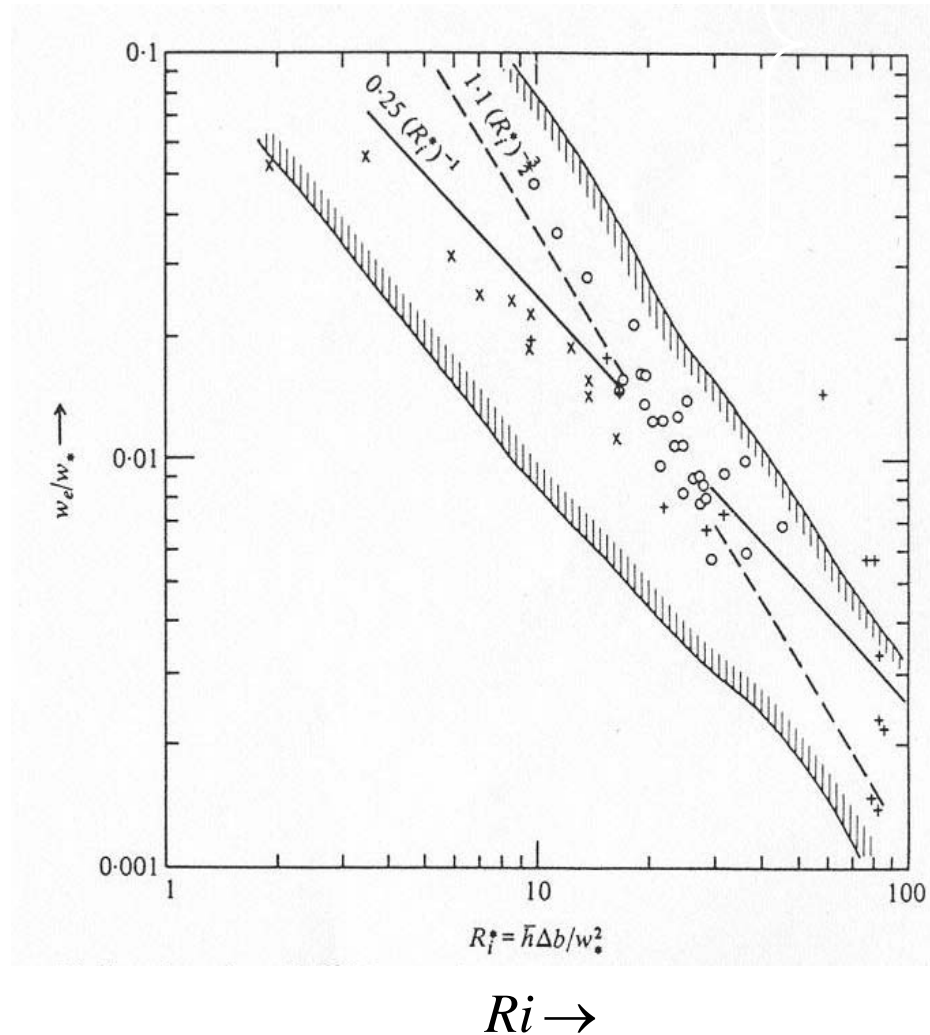
courtesy E. Fedorovich

Laboratory Experiments (thermal convection tank)

Entrainment rate

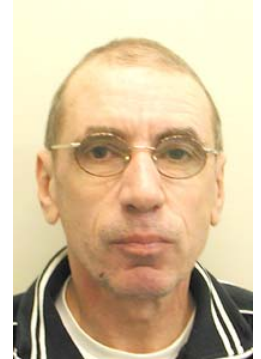
$$w_e = \frac{dz_i}{dt}$$

$$\uparrow \frac{w_e}{w_*}$$

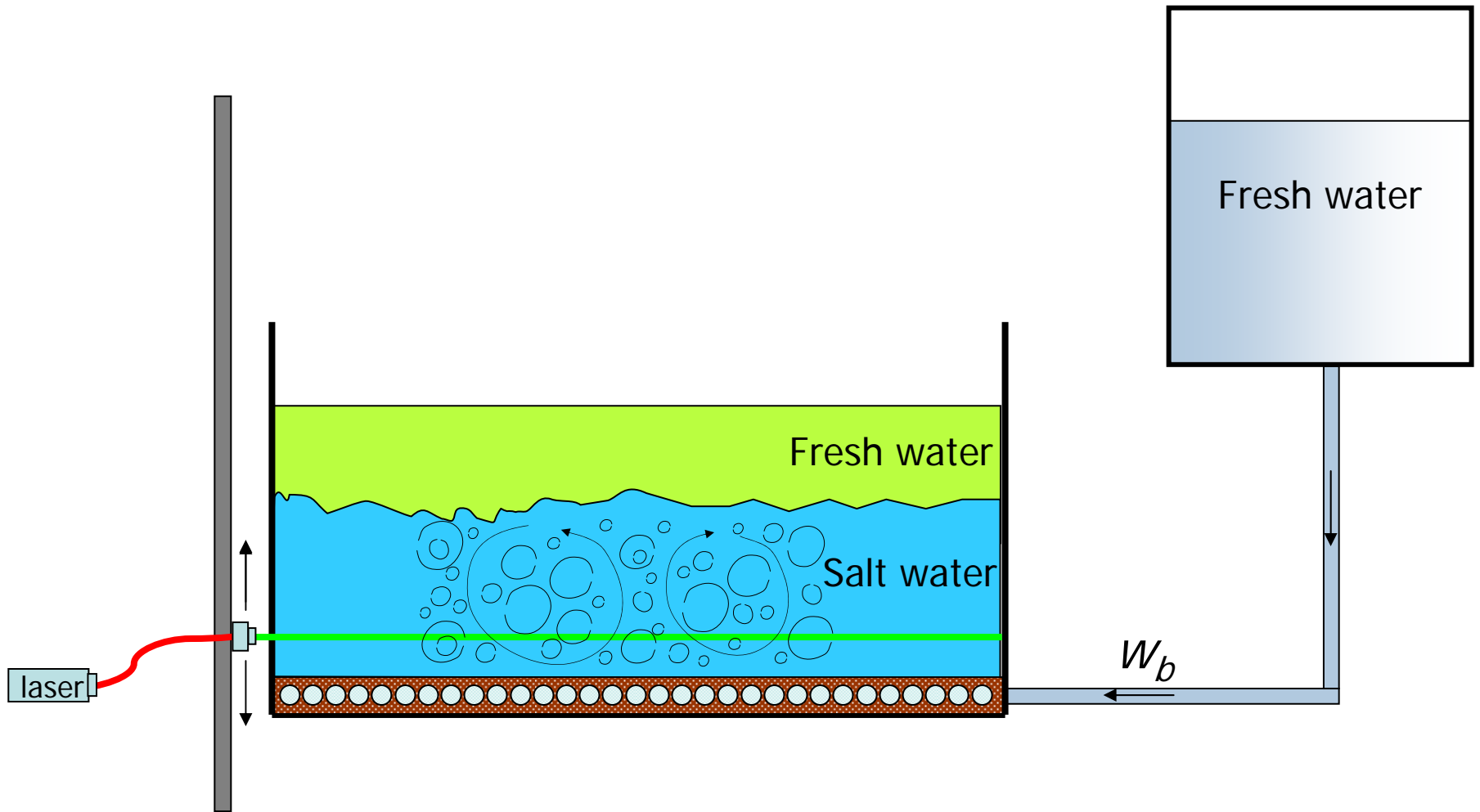


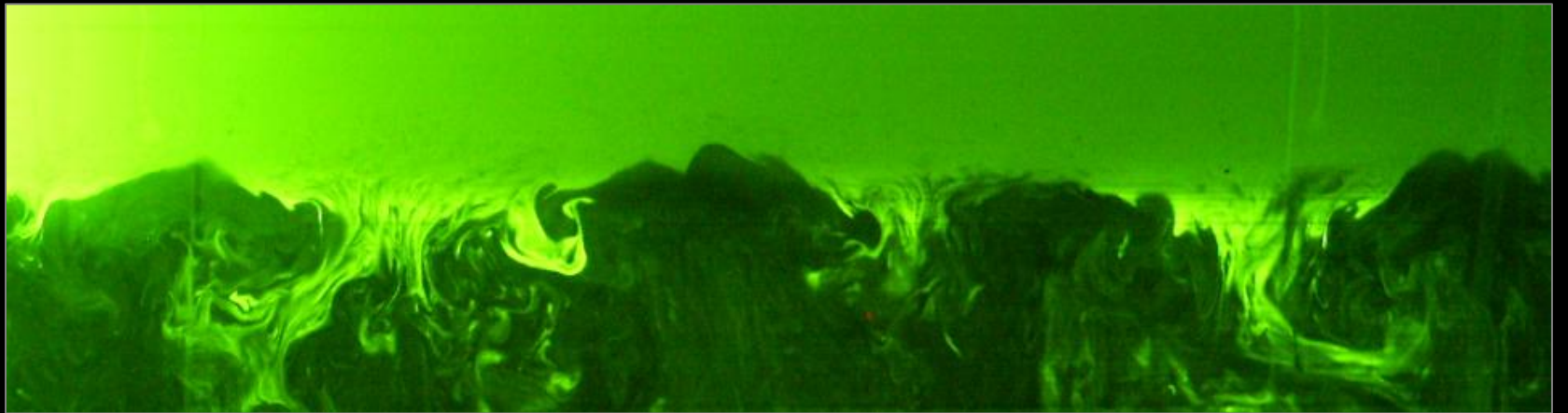
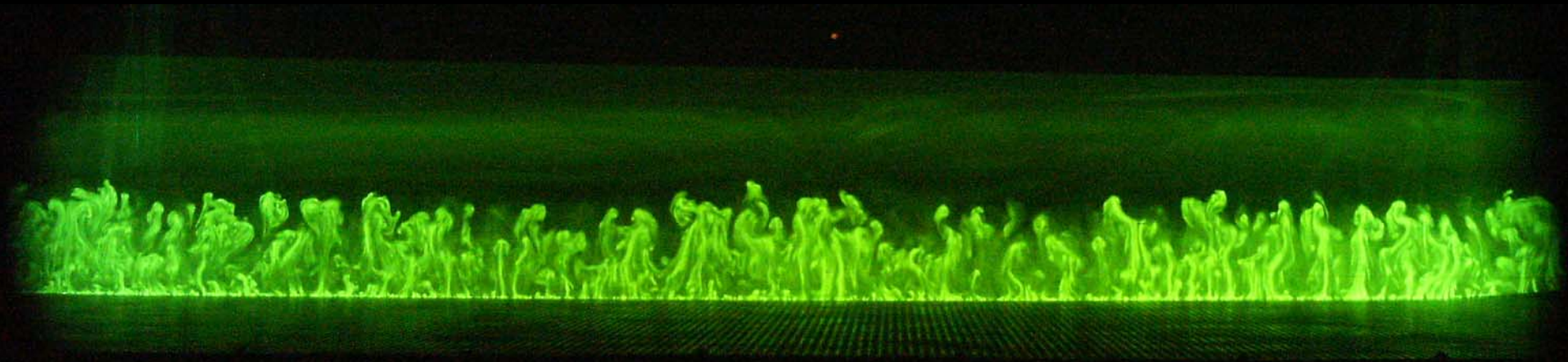
$$\frac{w_e}{w_*} = ARi^{-1}$$
$$A = 0.25$$

Deardorff, Willis and Stockton, JFM 1980

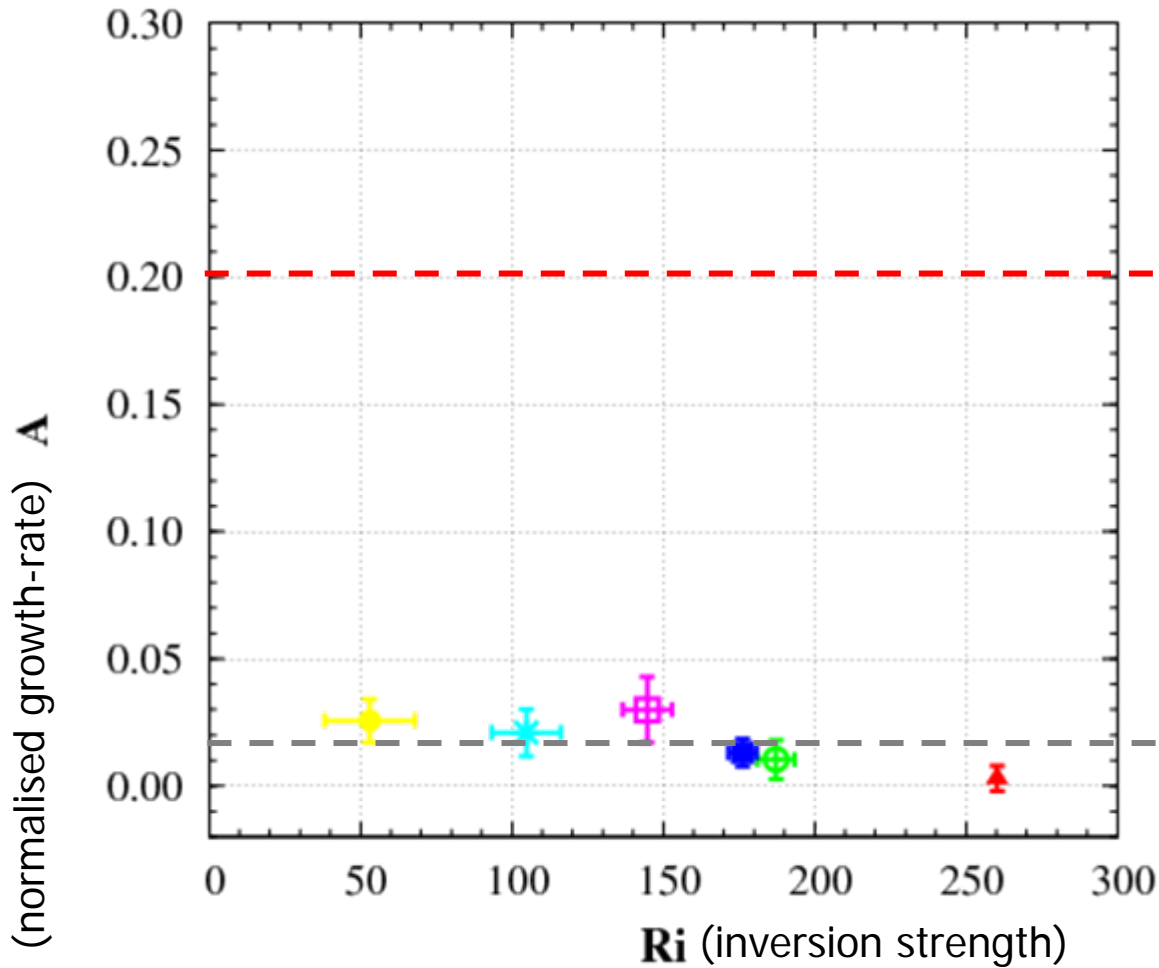


Experimental setup





experimental results



$A \approx 0.2$

factor 10 ??

$A \approx 0.02$

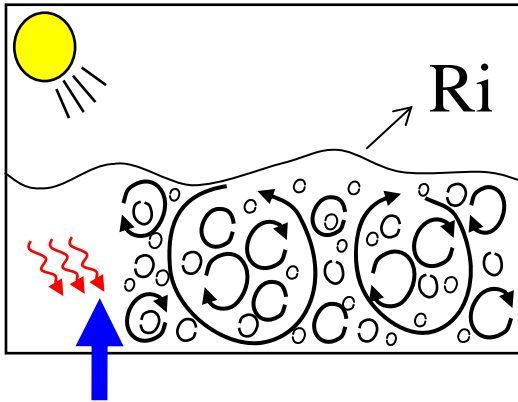
1970-1980

J. Deardorff

Controversy ...

J.S. Turner

Governing equations



range: ~1km to ~1mm

$$\frac{\partial u_i}{\partial t} + \frac{\partial u_j u_i}{\partial x_j} = \frac{1}{\text{Re}} \frac{\partial^2 u_i}{\partial x_j^2} - \frac{\partial p}{\partial x_i} + \theta \delta_{i3}$$

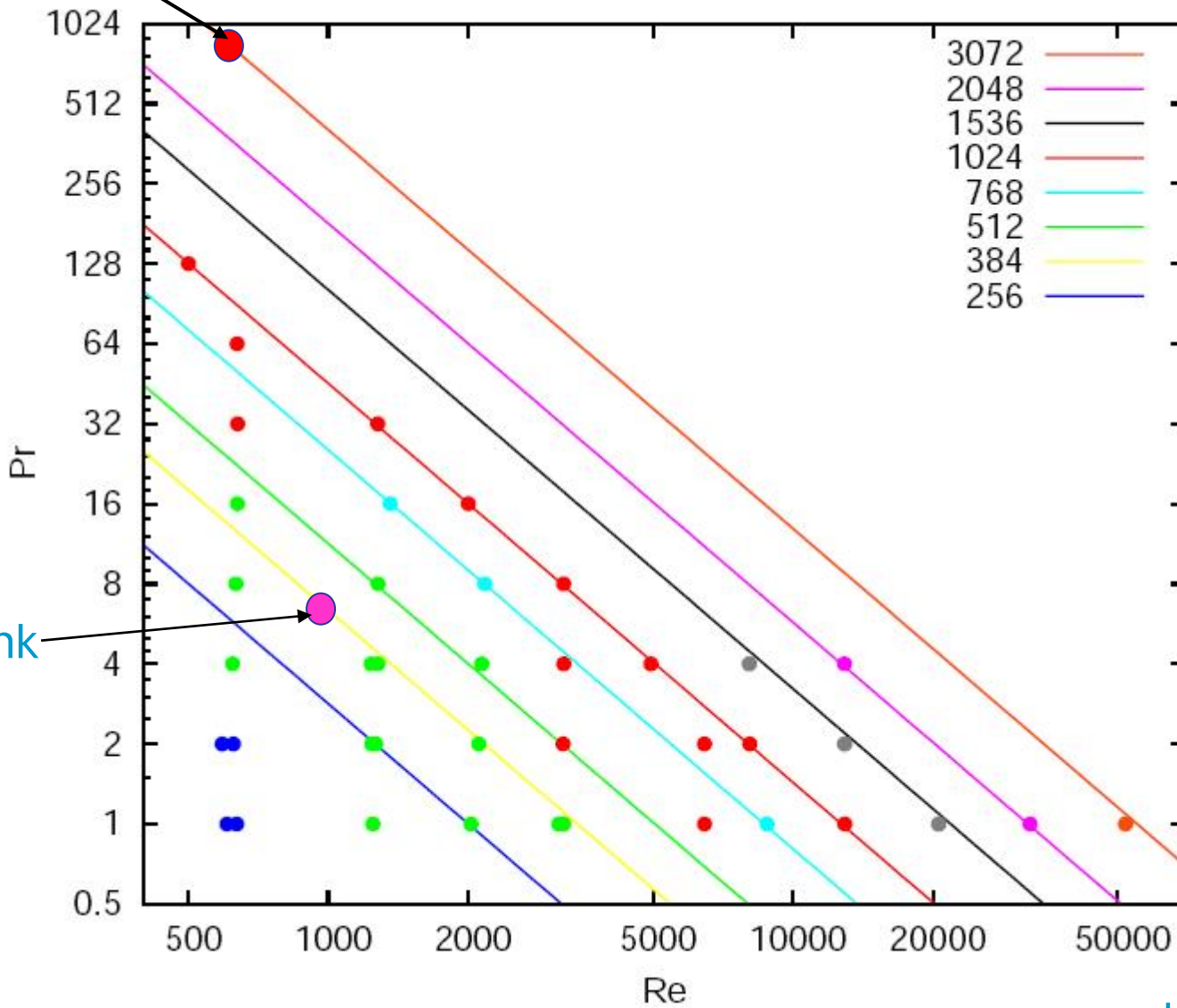
$$\frac{\partial \theta}{\partial t} + \frac{\partial u_j \theta}{\partial x_j} = \frac{1}{\text{RePr}} \frac{\partial^2 \theta}{\partial x_j^2}$$

		atmosphere	tank (heat)	tank (salt)
Reynolds number	$\text{Re} = \frac{w_* z_i}{\nu}$	$\text{Re} = 10^8$	$\text{Re} = 10^3$	$\text{Re} = 10^3$
Prandtl number	$\text{Pr} = \frac{\nu}{\kappa}$	$\text{Pr} = 1$	$\text{Pr} = 10$	$\text{Pr} = 10^3$
Peclet number	$\text{Pe} = \text{RePr}$	$\text{Pe} = 10^8$	$\text{Pe} = 10^4$	$\text{Pe} = 10^6$

DNS Cost $\sim N^4 \sim Re^3 Pr^2$

Re, Pr overview

saline tank



Deardorff tank

atmosphere



DEISA resource allocation: 2M cpu-hr

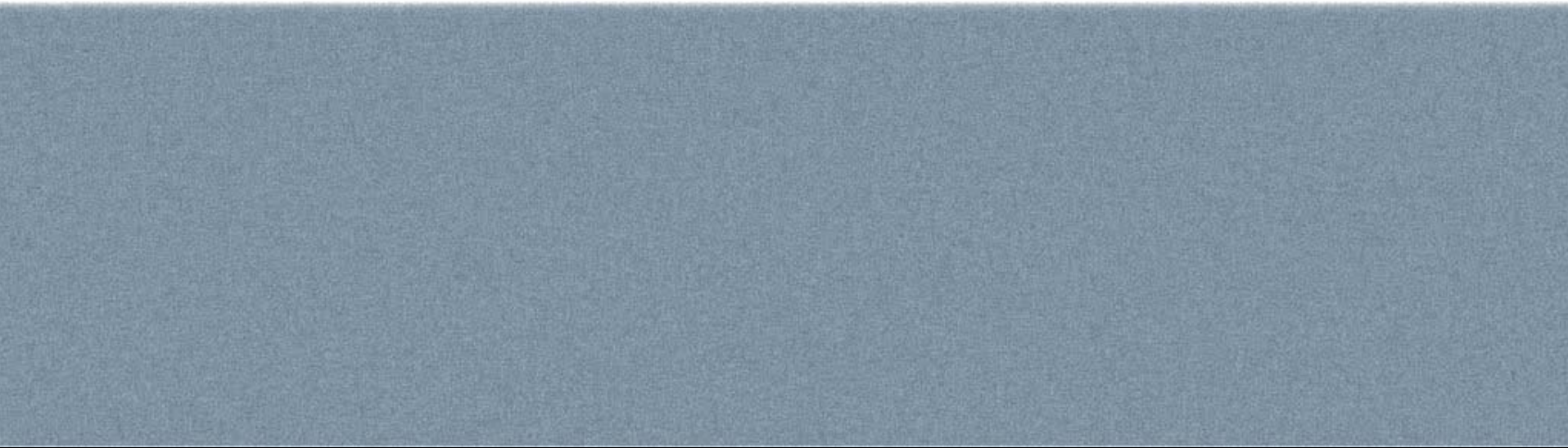
<i>Site</i>	<i>Architecture</i>	<i>cores used</i>	Grid
SARA	IBM Power 6	1024	1024 x 1024 x 768
CINECA	IBM BCX/5120	2048	2048 x 2048 x 1024
LRZ	SGI Altix 4700	3072	1536 x 1536 x 768
Juelich	Bluegene	32,768	3072 x 3072 x 1536

$$N_x = N_y = 2048, \quad N_z = 1024, \quad p = 2048$$

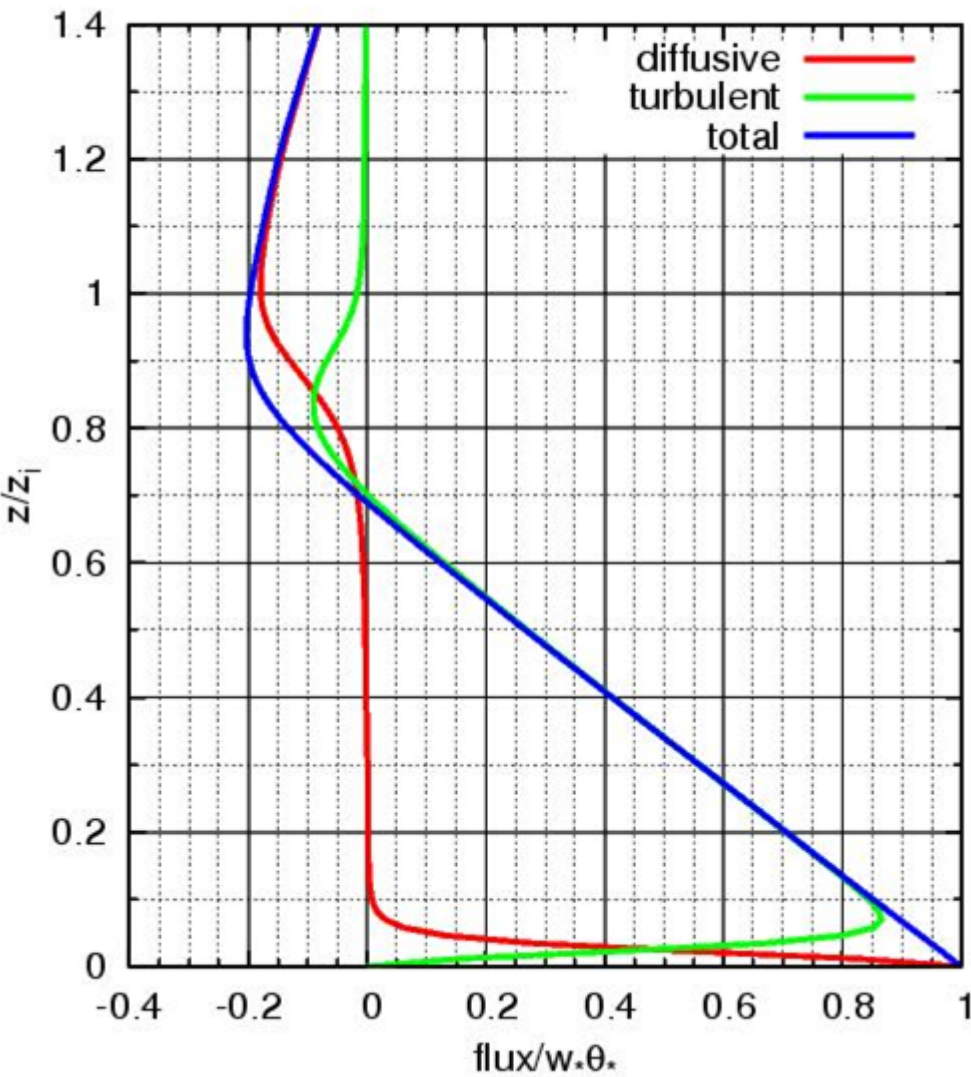
$$L_x = L_y = 3072m, \quad L_z = 1280m$$

$$\text{Re} = 30,000 \quad \text{Pr} = 1$$

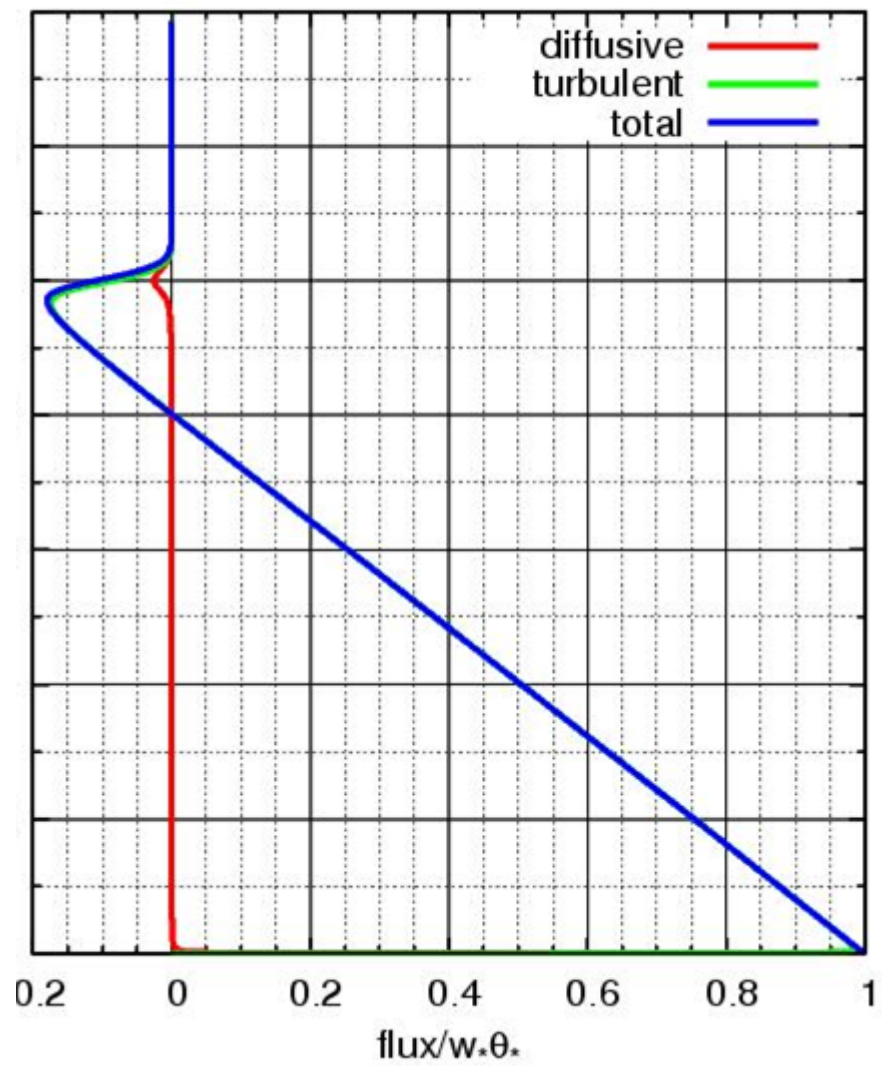
(potential) Temperature animation



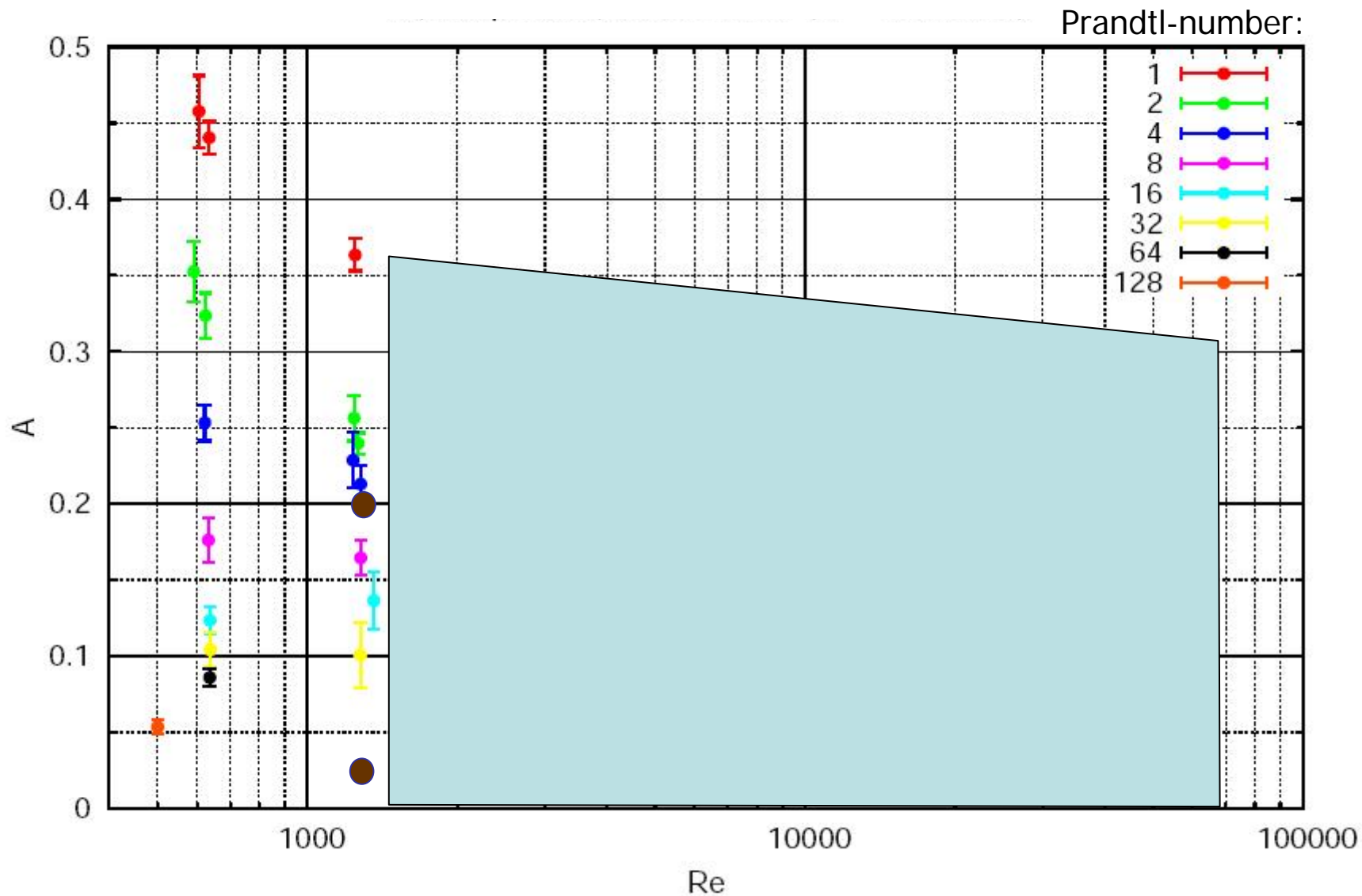
theta flux: Pr = 1, Re = 1260



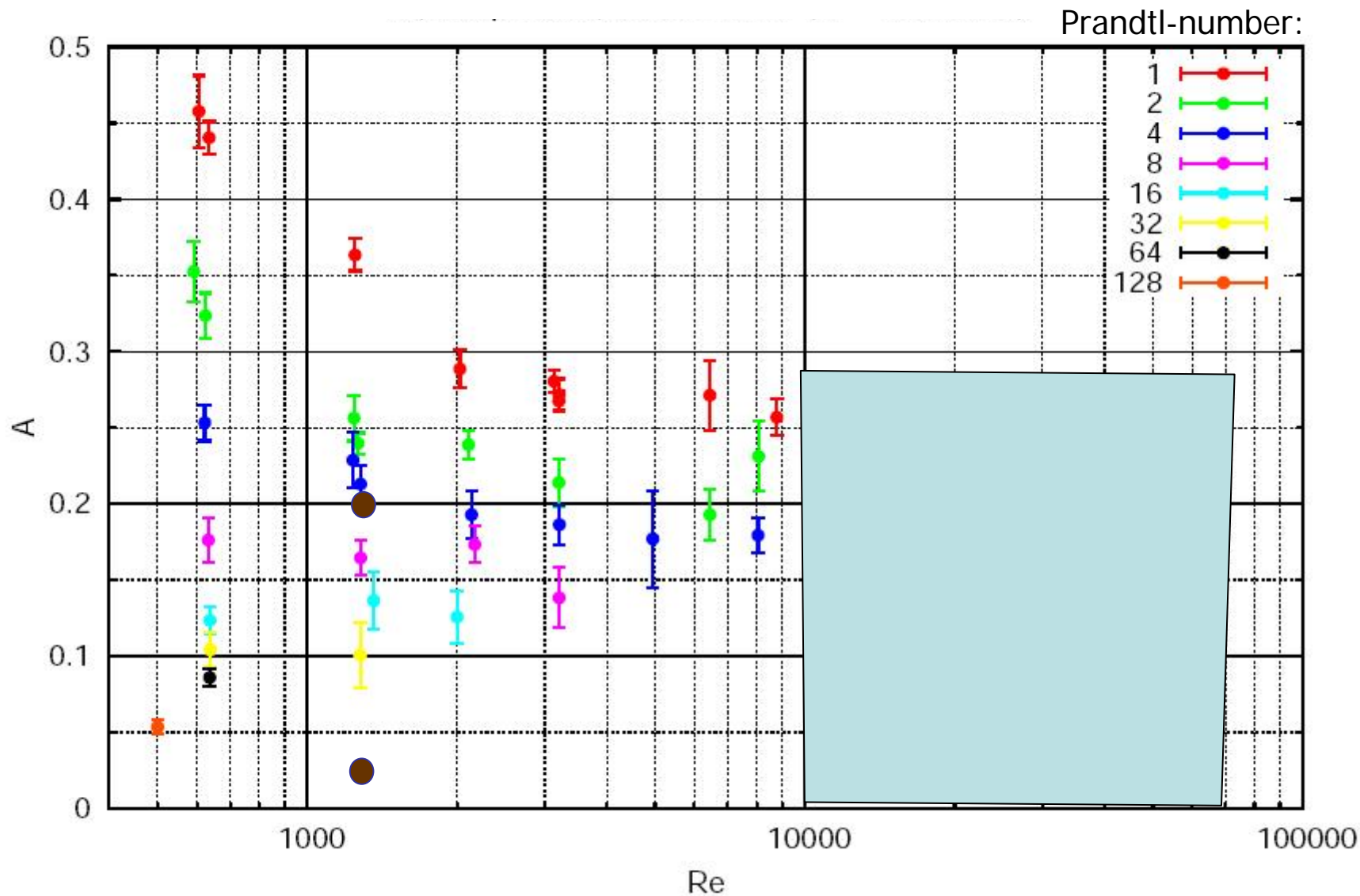
theta flux: Pr = 1, Re = 50,000



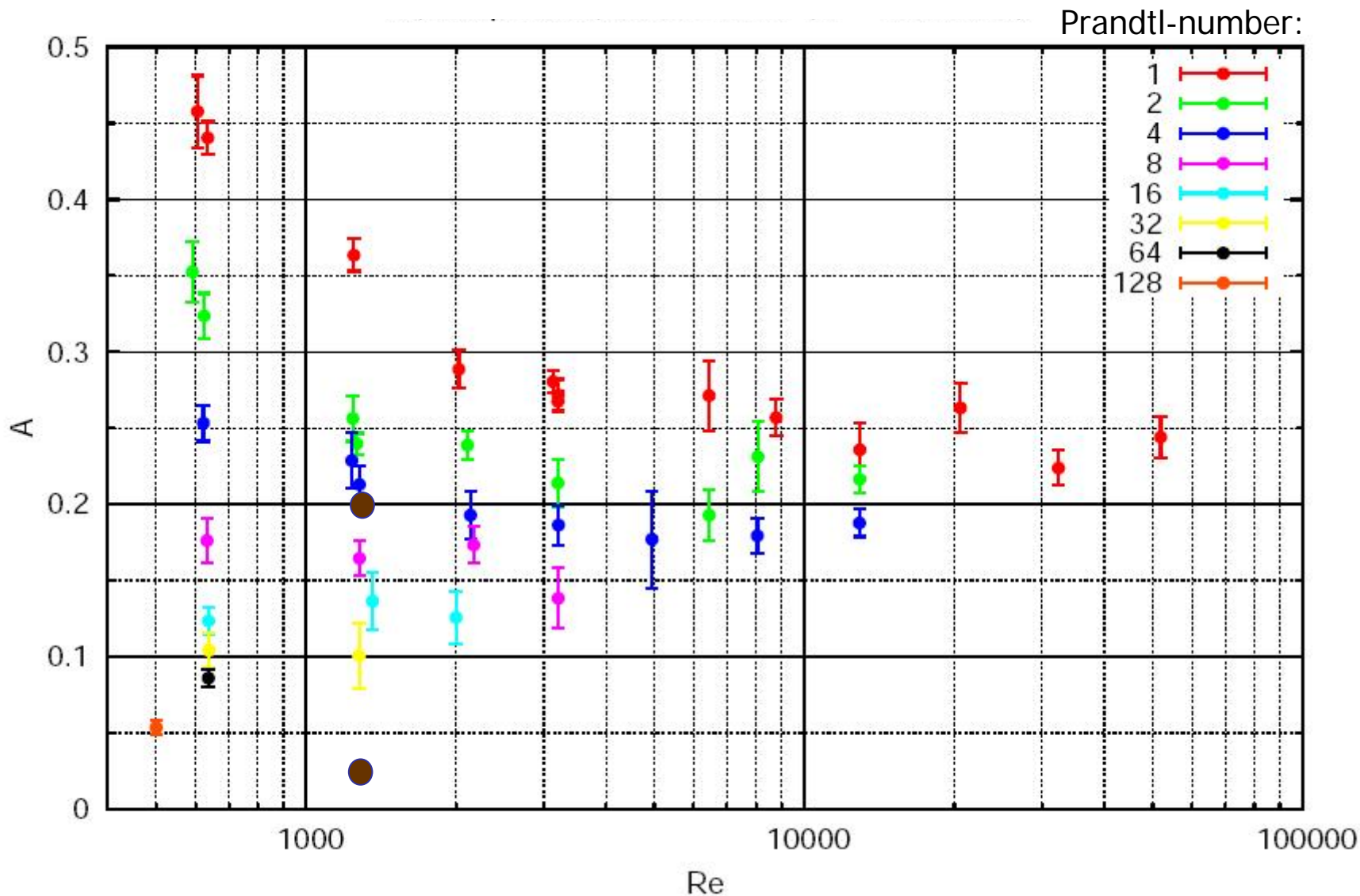
The importance of large computations



The importance of large computations

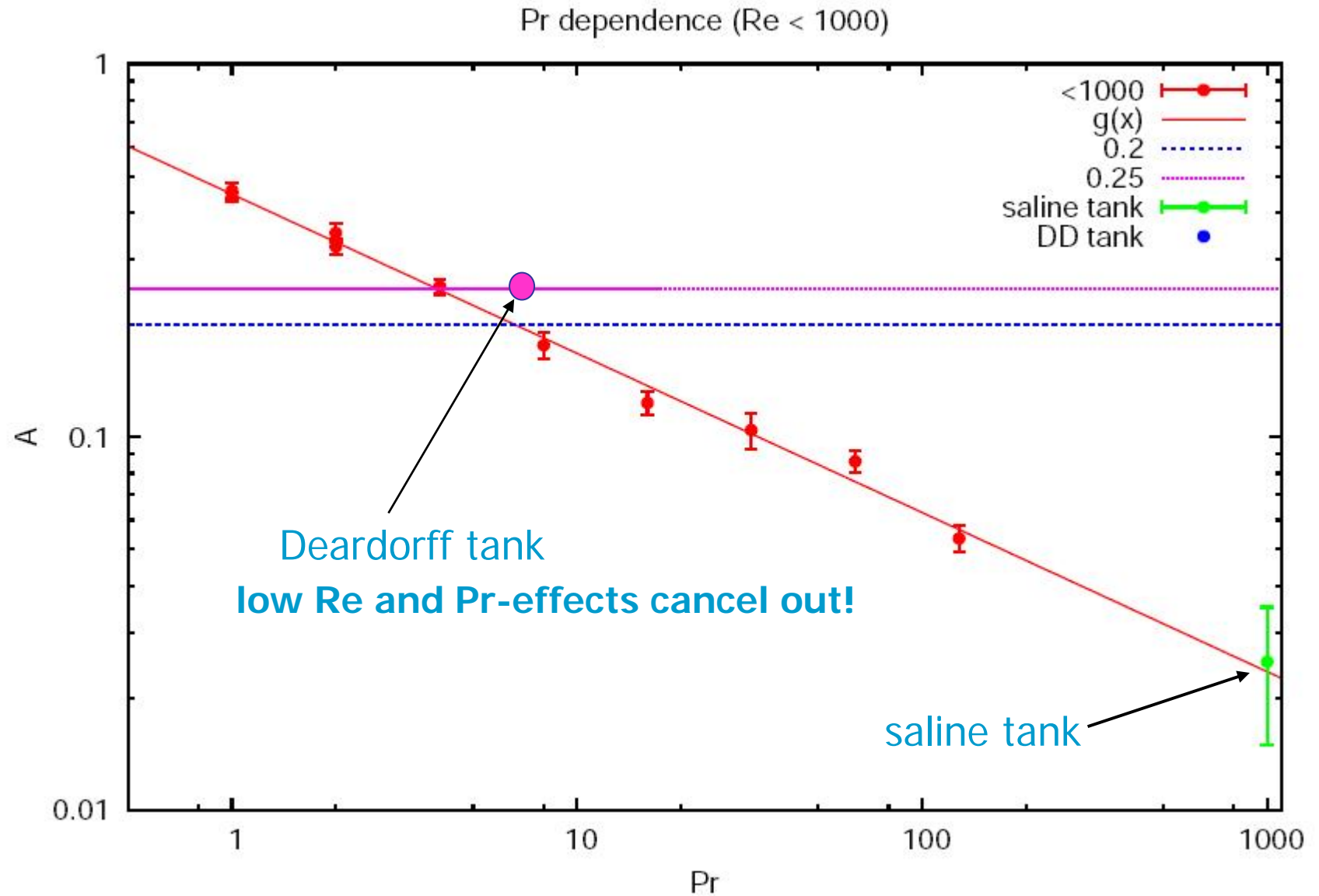


The importance of large computations

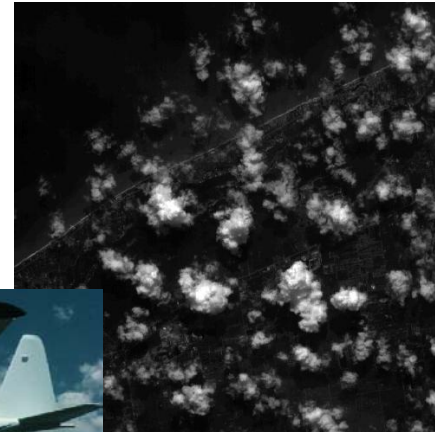
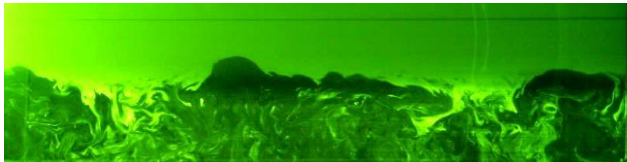
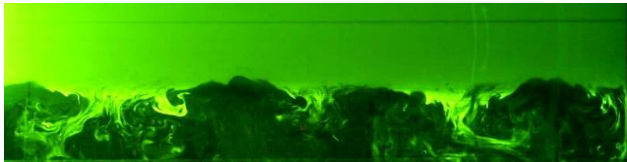
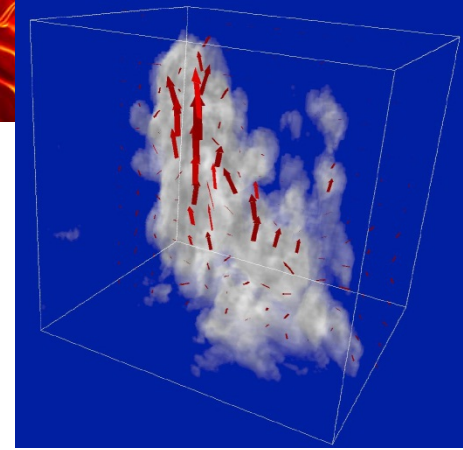
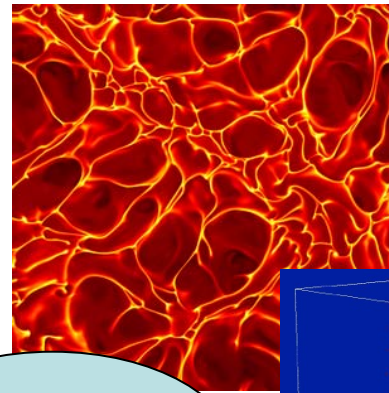
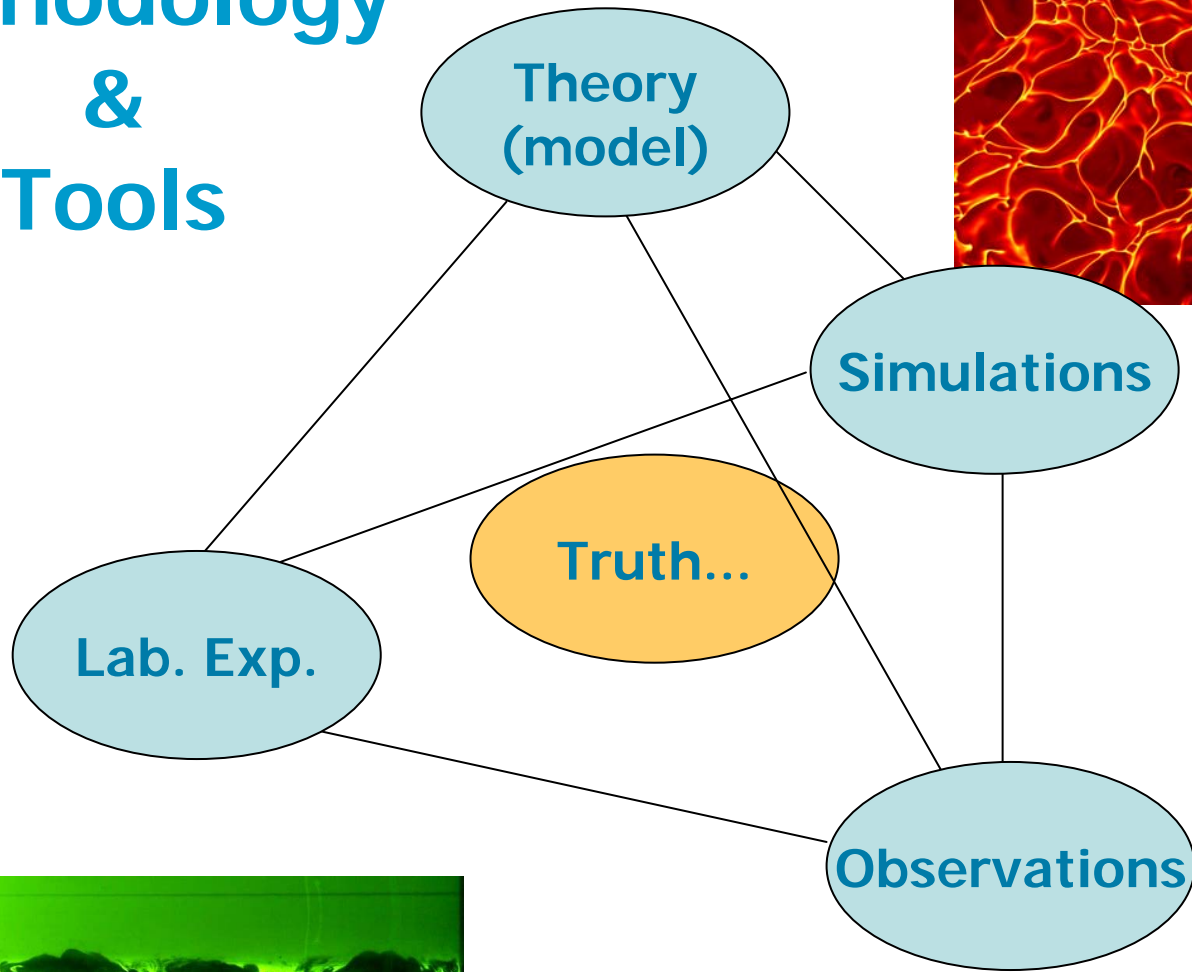


Re number must be really large before fluid-properties can be neglected

Fortuity or talent?



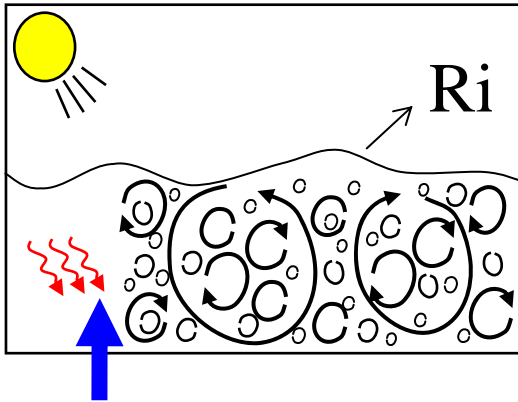
Methodology & Tools



Direct Simulation vs Understanding Turbulence

- DNS has now the ability to form the computational analog of laboratory experiments
- 'interesting' Reynolds numbers can be reached
- yields no immediate understanding
- ideal research tool (interactive)
- my impression: there seem a lack of hypotheses

Governing equations DNS



$$\frac{\partial u_i}{\partial t} + \frac{\partial u_j u_i}{\partial x_j} = \frac{1}{\text{Re}} \frac{\partial^2 u_i}{\partial x_j^2} - \frac{\partial p}{\partial x_i} + \theta \delta_{i3}$$
$$\frac{\partial \theta}{\partial t} + \frac{\partial u_j \theta}{\partial x_j} = \frac{1}{\text{RePr}} \frac{\partial^2 \theta}{\partial x_j^2}$$

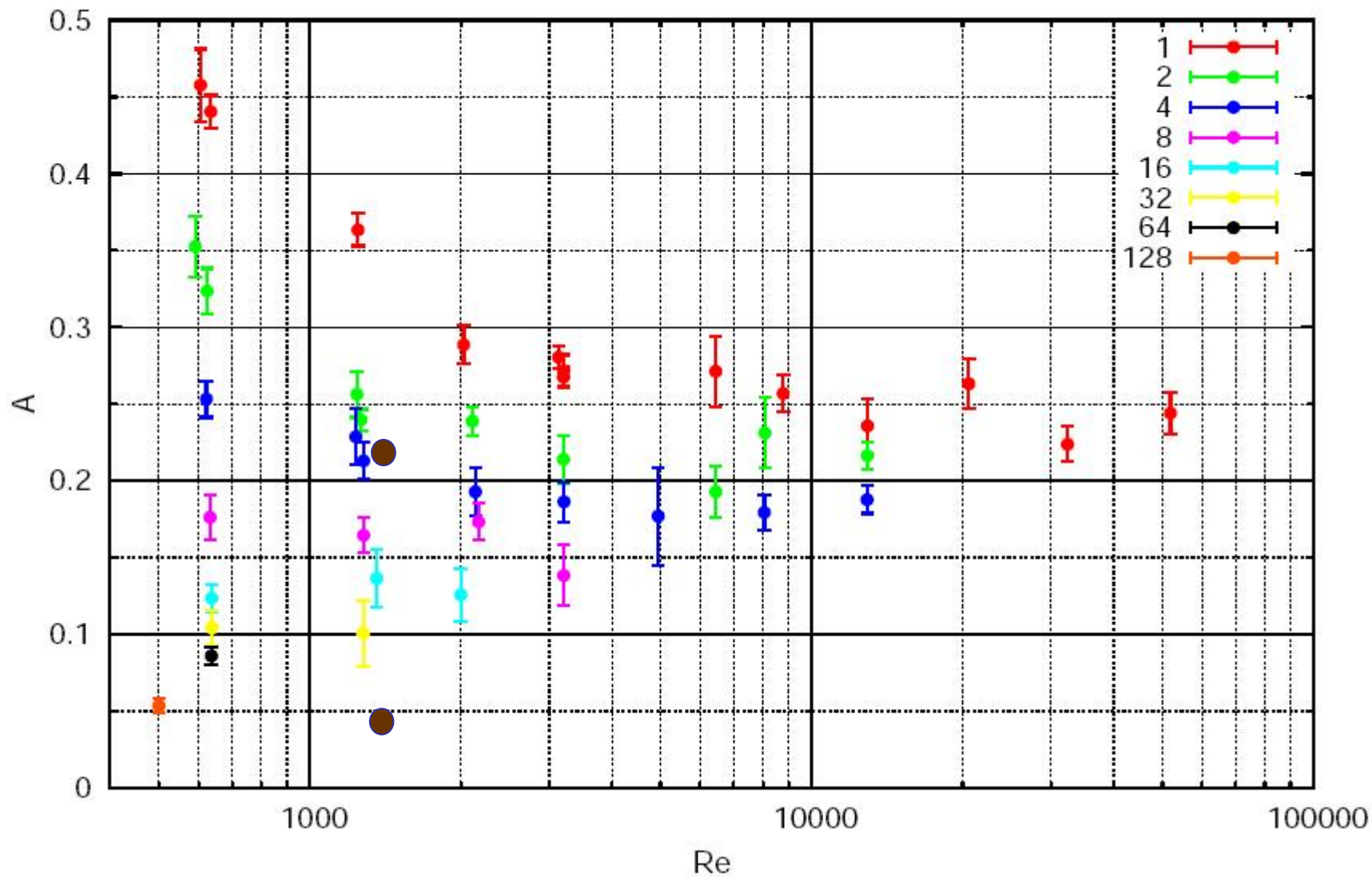
Compare to a Reynolds-stress turbulence model

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial \bar{u}_j \bar{u}_i}{\partial x_j} = \dots$$
$$\frac{\partial \bar{\theta}}{\partial t} + \frac{\partial \bar{u}_j \bar{\theta}}{\partial x_j} = \dots$$

wouldn't fit!

The Pr influence at large Reynolds numbers

Prandtl-number:

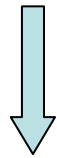


Large Reynolds/Peclet limits

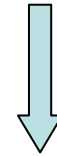
$$\frac{w_e}{w_*} = f(\text{Ri}, \text{Re}, \text{Pr}) \quad \text{or} \quad \frac{w_e}{w_*} = f(\text{Ri}, \text{Re}, \text{Pe})$$

$$\text{Re} \rightarrow \infty$$

$$(\text{Pe} \rightarrow \infty)$$



$$\frac{w_e}{w_*} = f(\text{Ri}, \text{Pr})$$

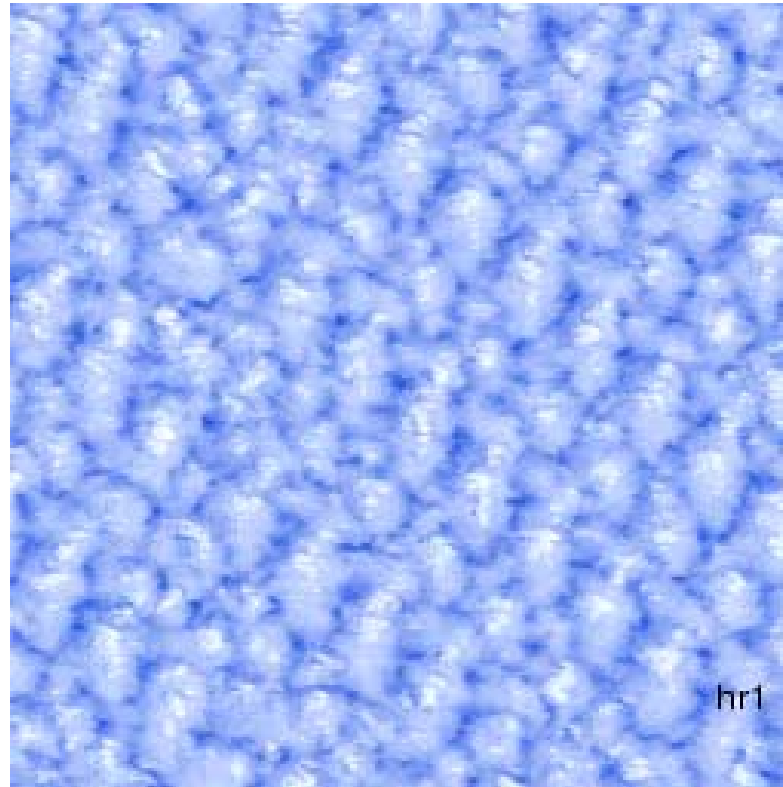


$$\frac{w_e}{w_*} = f(\text{Ri})$$

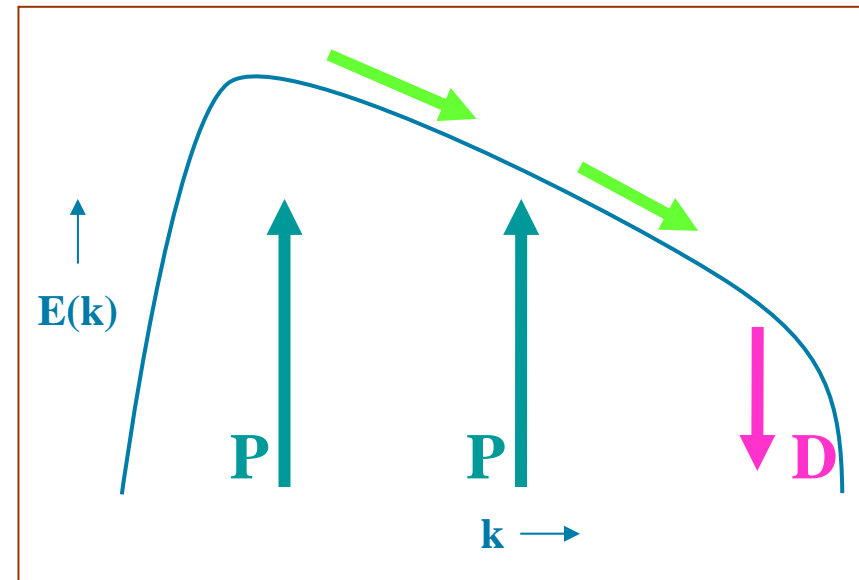
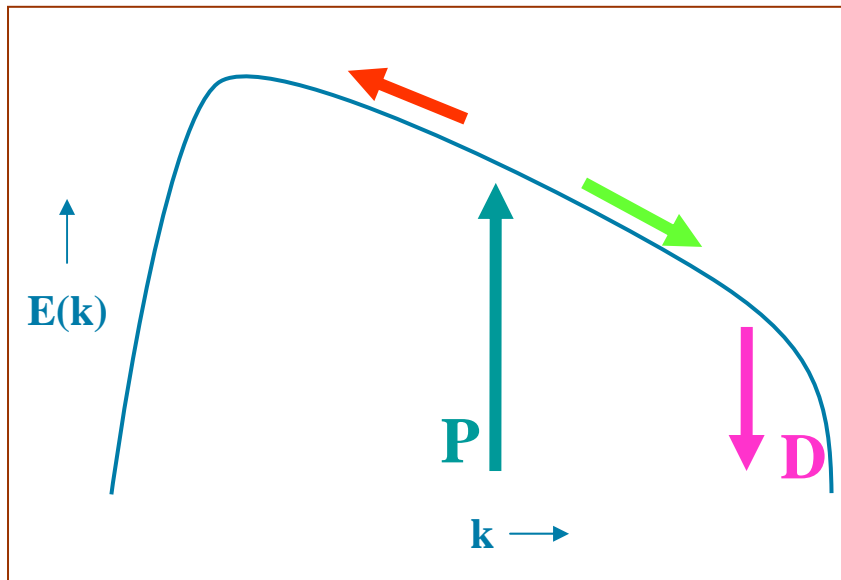
LES of Stratocumulus

$L = 25.6\text{km}$ $Dx = Dy = 100\text{m}$

$t = 1 \dots 16\text{hr}$, liquid water path

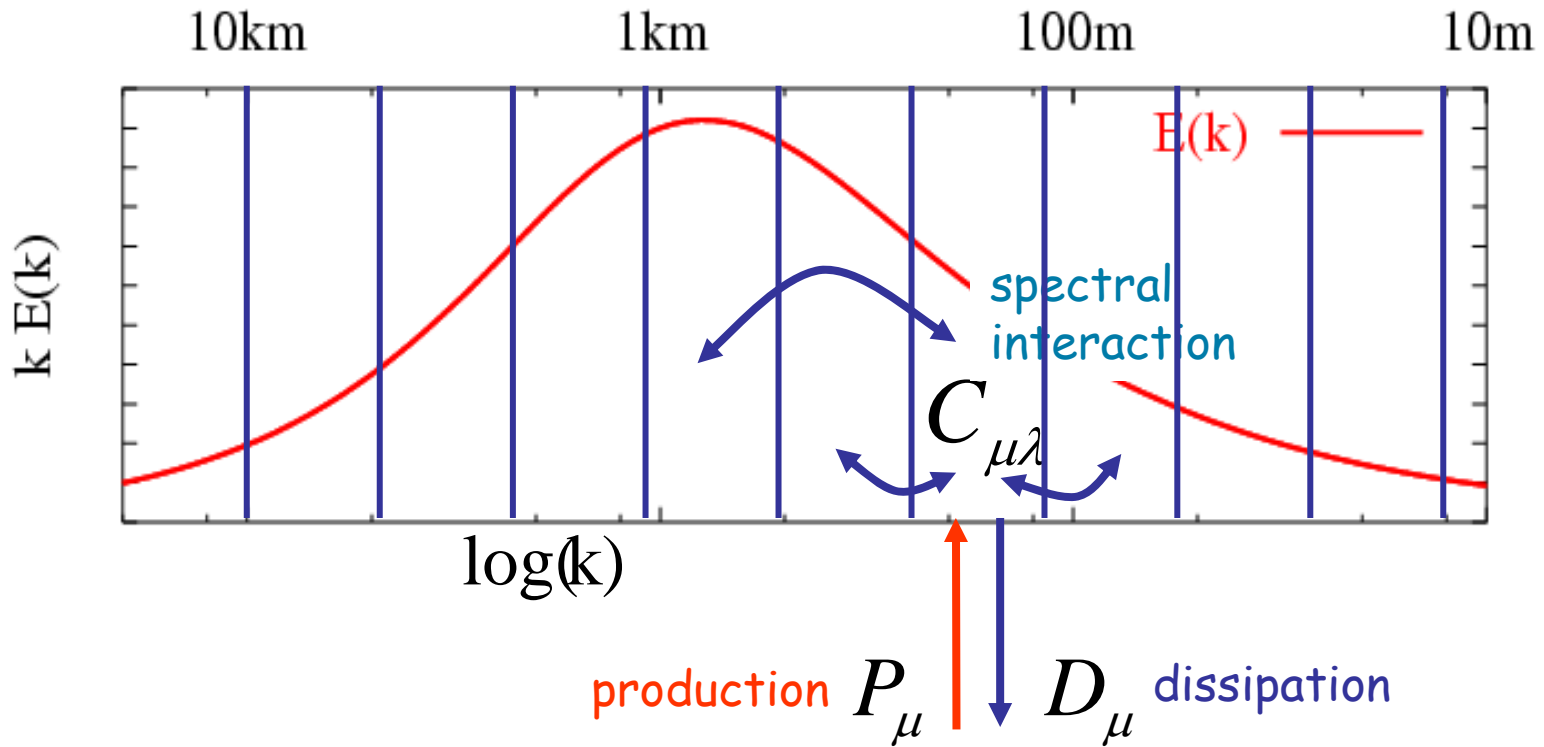


Inverse Cascade?



2-D or not 2-D: that's the question

Spectral variance budget



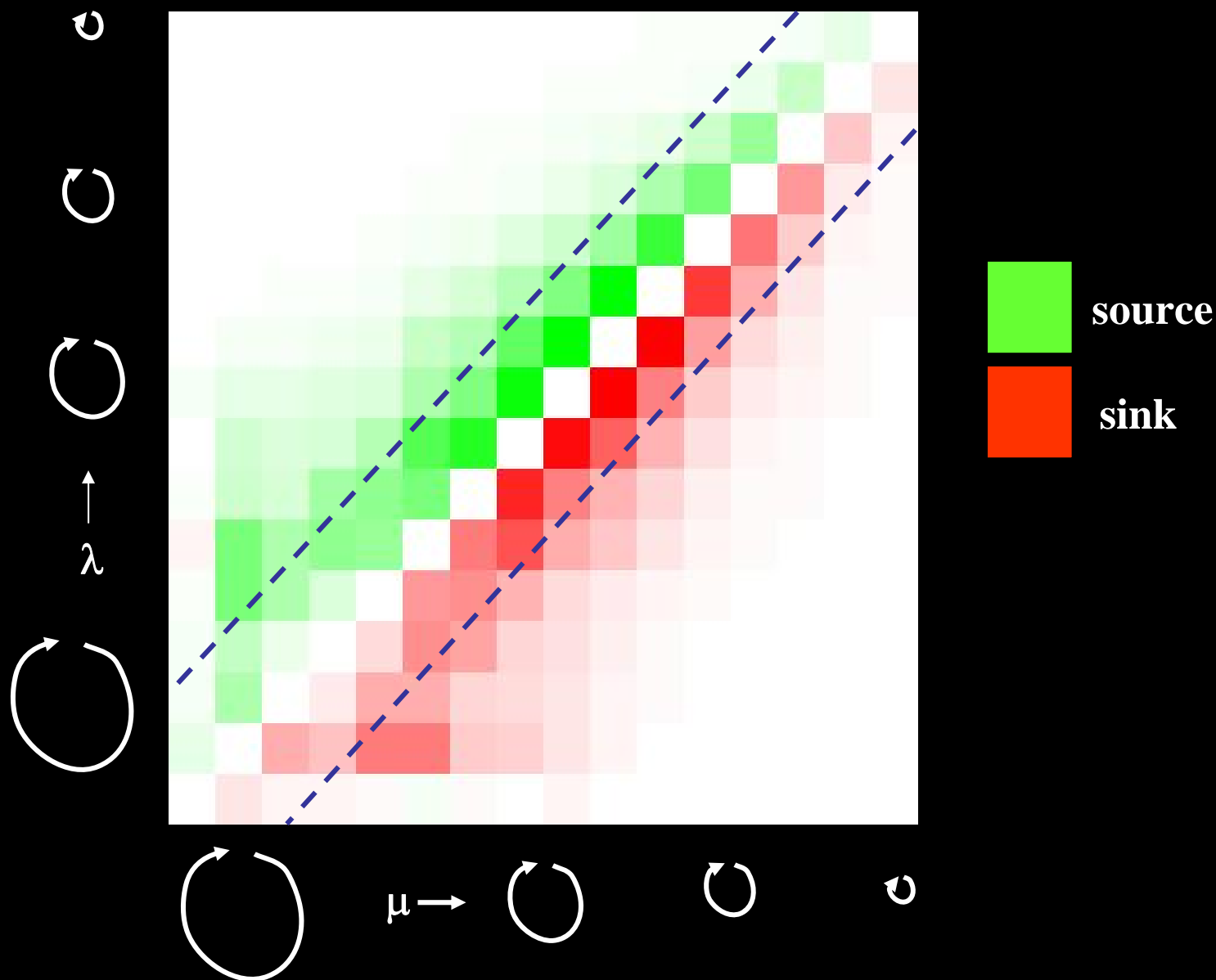
$$c = \bar{c} + c_1 + c_2 + \dots + c_n \quad \longrightarrow \quad \frac{d}{dt} \overline{c_{\mu}^2} = P_{\mu} - D_{\mu} + \sum_{\lambda} C_{\mu\lambda}$$

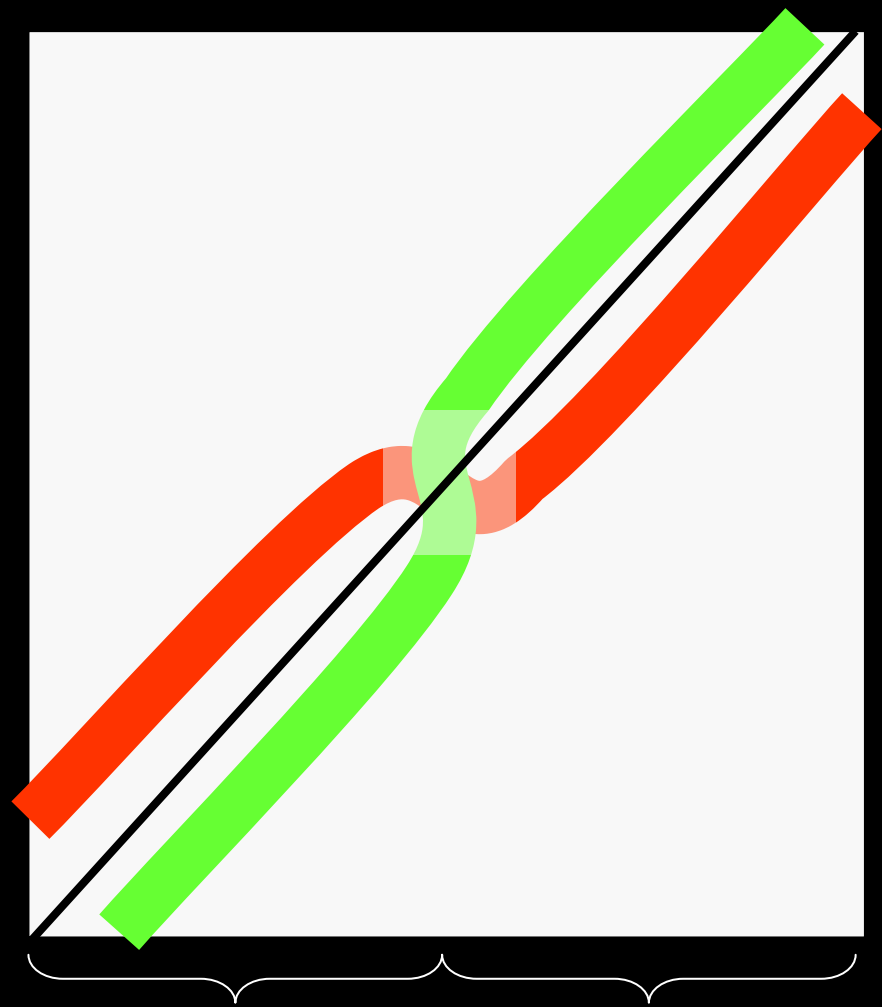
scale by scale variance budget

Scale Interaction Matrix $C_{\mu\lambda}$

16 sections

buoyancy field



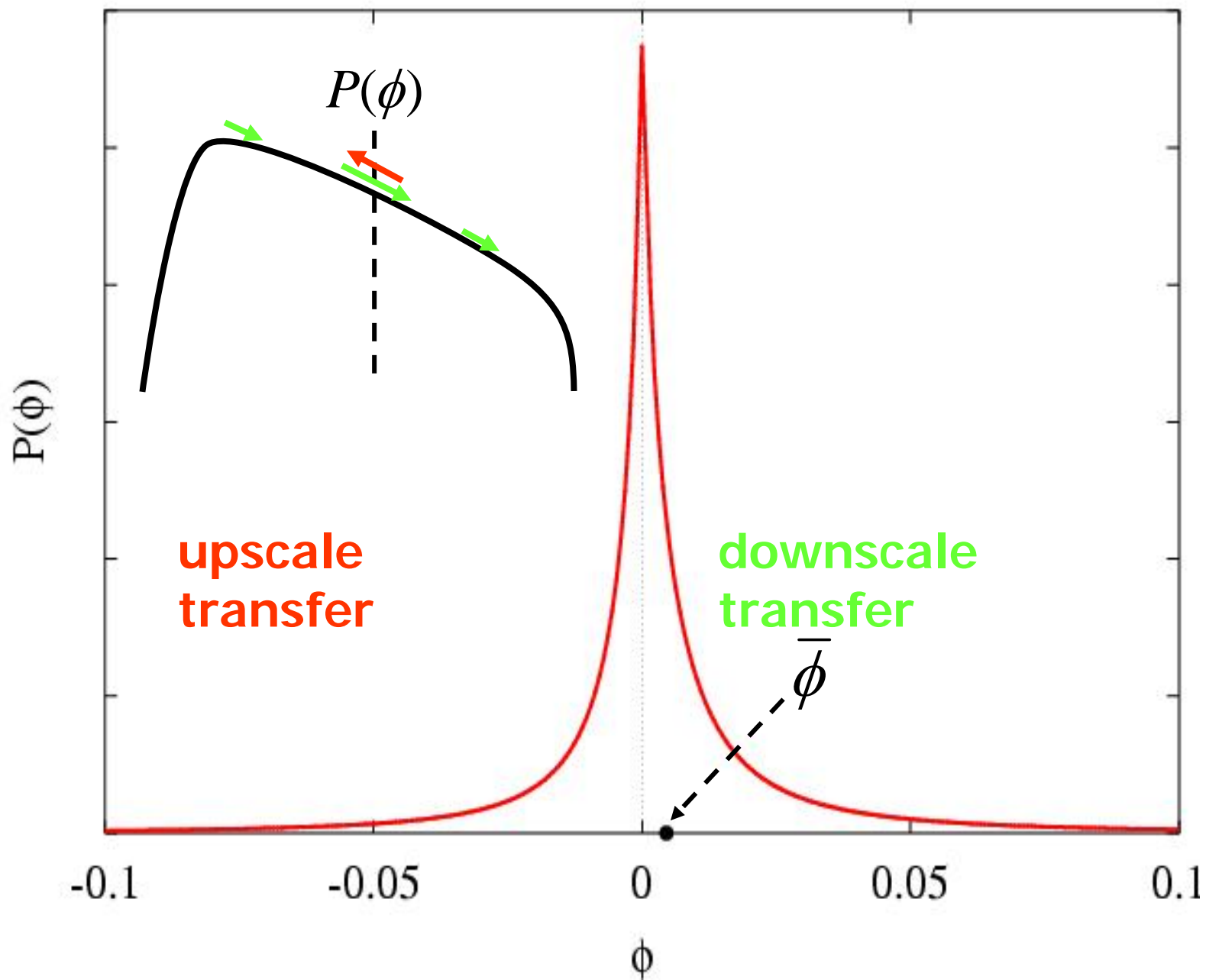


2D

3D

upscale cascade

downscale cascade



Concluding remarks

- **'Why modeling works' - does it really work?**
- **Direct Simulation will soon become the ideal research tool for turbulence (direct interaction)**
- **Are we capable of defining the most pressing hypotheses in turbulence?**